

## Research Article

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# Nonlocal state-space strain gradient wave propagation of magneto thermo piezoelectric functionally graded nanobeam

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**Abstract:** In this work, the state -space nonlocal strain gradient theory is used for the vibration analysis of magneto thermo piezoelectric functionally graded material (FGM) nanobeam. An analysis of FGM constituent properties is stated by using the power law relations. The refined higher order beam theory and Hamilton's principle have been used to obtain the motion equations. Besides, the governing equations of the magneto thermo piezoelectric nanobeam are extracted by developed nonlocal state-space theory. And to solve the wave propagation problems, the analytical wave dispersion method is used. The effect of magnetic potential, temperature gradient, and electric voltage in variant parameters are presented in graph.

**Keywords:** wave propagation, functionally graded materials, nonlocal strain gradient state-space theory, magneto piezoelectric nanobeam

## 1 Introduction

Functionally graded materials (FGMs) are a type of composites initiated by a group of Japanese scientists to control the volume fraction of the mixture of two or more materials. The nonlinear vibration of the piezoelectric nanobeams based on the nonlocal and Timoshenko theory,

the influence of the nonlocal parameter, temperature change, and external electric voltage on the size dependent nonlinear vibration characteristics of the piezo electric nanobeam are exposed [1]. Researchers [2] studied the natural frequencies along with mechanical and thermo electric vibration of piezoelectric nanobeams based on the nonlocal theory. Ebrahimi [3] reported the scattering of waves of FG nanobeam of viscoelastic nature. In the framework of third-order shear deformation theory [4], the vibration characteristics of functionally graded (METE-FG) nanobeams were analyzed. And the free vibrations of FG nano plates resting on elastic foundation *via* Hamilton principle was dealt in detail [5]. Alibeigi *et al.* [6] introduced the buckling retaliation of nanobeams on the basis of the Euler–Bernoulli beam model with the von Kármán geometrical nonlinearity. Bending of flexo electric magneto-electro-elastic (MEE) nanobeams lying over Winkler–Pasternak according to nonlocal elasticity theory has been studied [7]. Several studies were conducted on [8–10] hygro-thermal loading, the bending analysis of magneto-electro piezoelectric nanobeams system, dynamic analysis of smart nanostructures, and frequency analysis of thermally post buckled FGM thin beams. Stress-driven vs strain-driven elastic nanobeams have been discussed *via* integral elasticity [11,12]. Using the kinematic model, Kiani and Eslami [13] reported the buckling of beams made of FG under different types of thermal loading. The propagation of wave of infinite functionally graded plate in thermal environment was reported by Sun and Luo [14]. A consistent refined HSDT is designed to probe the free vibration of GF plates on elastic foundation and the influence of boundary condition on the natural frequency [15]. The different working conditions of the nano sized elements were studied by Thai *et al.* [16]. By considering the nonlocal elasticity [17], a prediction has been made that the essential behaviors of the nanostructures cannot be same as the macro scale structures. The Euler–Bernoulli beam theory was used to study the bending analysis of microtubules (MT) by Eringen [18]. Based on Euler–Bernoulli beam theory, the bending analysis of MT

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was studied using the method of Differential Quadrature (DQ) by Civalek and Demir [19]. Using finite element method (FEM) [20], the nonlinear bending in nanobeams were discussed. Reddy and El-Borgi [21] investigated the dispersion of waves with the effects of surface stresses in smart piezoelectric nanoplates. Bi-directional FGM nanobeams with the characteristics of bending, buckling, and vibrational nonlocal elements were concentrated in some of the previous studies [22–24]. The natural frequency variation located on a viscoelastic sheet was surveyed by using the nonlocal theory [25]. The size-dependent elements of beam were analyzed by Ebrahimi and Barati [26]. Nonlocal elasticity and its running conditions are discussed in details in the literature [27]. Based on the nonlocal strain gradient theory (NSGT) [28], the thermo-mechanical buckling problem of graphene sheets was proposed. Stiffness, softening, and hardening effect of FG beam were studied by Li *et al.* [29]. Solving the wave dispersion problem of nanoplates was accomplished by Ebrahimi and Dabbagh [30] with the application of infusing NSGT and surface-related elasticity for responsive piezoelectric materials. With the small-scale effect, the free vibration of 3D FGM Euler–Bernoulli nanobeam was studied by Hadi *et al.* [31]. Alibeigi *et al.* [32] exposed the buckling response of a nanobeam on the basis of the Euler–Bernoulli beam model using a couple stress theory under various types of thermal loading and an electrical and magnetic field. Timoshenko beam theory was investigated by Ke and Wang [33] with the rise in uniform temperature, magnetic potential, and external electric potential *via* nonlocal form to MEE vibrations. Bending of MEE nanobeam was studied in detail by Ebrahimi *et al.* [34]. Along with that, Ebrahimi *et al.* [35] investigated the bending of MEE nanobeams relating the nonlocal elasticity theory under hygro-thermal loading embedded in Winkler–Pasternak foundation. The size dependent problems using nonlocal elasticity theory, nonlocal couple stress theory, and shear deformation theory were reported [36,37]. Ebrahimi *et al.* [38] discussed the effects of various parameters on the wave dispersion characteristics of size-dependent nanoplates. The thermal effects on the buckling and free vibration of the FG nanobeams is documented well in the literature [39]. Ebrahimi and Barati [40] discussed the damping vibration characteristics of the hygro-thermally affected FG viscoelastic nanobeams. The thermal effect on buckling and free vibration characteristics of size-dependent Timoshenko nanobeams, and the free vibration of curved FG nano size beam in thermal environment been discussed in the literature [41,42]. The buckling and vibration properties of sandwich FG beams were studied by Vo *et al.* [43]. Jalaei *et al.* [44] studied the thermal and magnetic effects on the FG Timoshenko nanobeam. Studies over the hygro-thermal

wave characteristic of nanobeam of an inhomogeneous material with porosity under magnetic field is notable [45].

Hence, this work shows the wave propagation analysis of FG nanobeam with the help of a nonlocal state-space strain gradient viscoelasticity. The magneto thermo material properties of the nanobeam also graded and implemented *via* power law relations and the motion equations are deduced through the Hamilton’s principle. Furthermore, the dispersion for computed external electric voltage, magnetic effect, and the gradient of temperature are presented with graphical solution.

## 2 Mathematical formulations

Based on the state-space nonlocal strain gradient theory, the length  $L$ , width  $b$ , and thickness  $h$  of a viscoelastic FG nanobeam has been investigated (Figure 1). The two parts of constituent FGM are composed of ceramic part and metallic part. The components of FGM are considered to be temperature dependent to evolve a realistic viscoelastic study.

In this section, power-law relations have been used to compute the properties. To calculate the temperature variable towards the thickness direction, the volume fraction of each phase must be calculated by using the power-law model. Hence, the volume fraction of ceramic partis given as follows:

$$V_c = \left( \frac{z}{h} + \frac{1}{2} \right)^p, \quad (2.1)$$

where the thickness  $h$  and the exponential power law  $p$  probe each phase in the material with its distribution of properties. By considering any desired material property at the local temperature

$$P = P_0(P_{-1}T^{-1} + 1 + P_1T + P_2T^2 + P_3T^3), \quad (2.2)$$

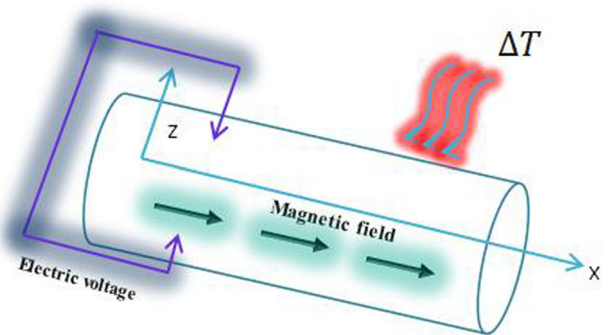


Figure 1: Vibration analysis of FG nanobeam.

where  $P_0, P_{-1}, P_1, P_2, P_3$  are the coefficients of material phases. The volume fraction of the metallic phase gives the volume fraction of the ceramic phase by  $V_m = 1 - V_c$ .

According to Eringen's nonlocal theory, the stress state at a point inside a body is a function of the strains at all points in the neighboring regions. The basic equations with zero body force can be defined as follows:

$$\begin{aligned}\sigma_{ij} &= \int_V \alpha(|y' - y|, \tau) [C_{ijkl} \varepsilon_{kl}(y') - e_{mij} E_m(y') - \Omega_n(y') \\ &\quad - C_{ijkl} \alpha_{kl} \Delta T] dV(y') \\ D_i &= \int_V \alpha(|y' - y|, \tau) [e_{ikl} \varepsilon_{kl}(y') - E_m(y') + \Omega_n(y') \\ &\quad - \Delta T] dV(y') \\ B_i &= \int_V \alpha(|y' - y|, \tau) [\varepsilon_{kl}(y') + E_m(x') + \chi_{ni} \Omega_n(y') \\ &\quad - \lambda_i \Delta T] dV(y'),\end{aligned}\quad (2.3)$$

where  $\sigma_{ij}$ ,  $\varepsilon_{ij}$ ,  $D_i$ ,  $E_i$ , respectively, represent the stress, strain, electric displacement, and electric field and  $B_i$ ,  $\Omega_i$  are the magnetic induction and magnetic field.  $\alpha_{kl}$  and  $\Delta T$  stand for thermal expansion and temperature difference.  $C_{ijkl}$ ,  $e_{mij}$ , and  $\chi_{ij}$  are the elastic, piezoelectric, and magnetic constants, respectively, and  $\tau = e_0 a/l$  defines the scale coefficient,  $e_0$  is the material constant, and  $a$  and  $l$  are the characteristic length in the internal and external sides.

The governing equations of the nanobeams are obtained by an accurate kinematic theory. Higher order shear deformation theory also reveals stress-strain changes in solid bodies. From the previous study [8], we can take the refined shear deformable beam's displacement as follows:

$$\begin{aligned}\bar{\Pi}_x(x, z, \tilde{t}) &= \bar{\Pi}(x, \tilde{t}) - \frac{\partial w_b(x, \tilde{t})}{\partial x} z - \frac{\partial w_s(x, \tilde{t})}{\partial x} f(z) \\ \bar{\Pi}_z(x, z, \tilde{t}) &= w_b(x, \tilde{t}) + w_s(x, \tilde{t}),\end{aligned}\quad (2.4)$$

where  $\Pi$ ,  $w_b$ , and  $w_s$  are the longitudinal displacement, bending, and shear components of the transverse displacement. Furthermore, in order to distribute the shear strain,  $f^*(z)$  is the shape function which is designed as follows [4]:

$$f^*(z) = \frac{he^z}{h^2 + \pi^2} \left[ \pi \sin\left(\frac{\pi z}{h}\right) + h \cos\left(\frac{\pi z}{h}\right) \right] - \frac{h^2}{h^2 + \pi^2}, \quad (2.5)$$

To capture the shear strain and stress, the deformed structural cross section is uncertain with this function. At free surfaces, it is required to satisfy the assumption of shear strain nonexistence. By continuum infinitesimal strain tensor, the nonzero strains can be measured as follows:

$$\begin{aligned}\varepsilon_{xx} &= \frac{\partial \Pi}{\partial x} - z \frac{\partial^2 w_b}{\partial x^2} - f(z) \frac{\partial^2 w_s}{\partial x^2} \\ \gamma_{xz} &= g(z) \frac{\partial w_s}{\partial x},\end{aligned}\quad (2.6)$$

where  $g(z) = 1 - \frac{df(z)}{dz}$ .

## 2.1 Motion equations

In accordance with Hamilton's principle, the extended Lagrangian can be given as follows:

$$\dot{L} = \dot{\Pi} - \dot{T} + \dot{V}, \quad (2.7)$$

So, the Hamilton's principle can be given as follows:

$$\delta \int_{\tilde{t}_1}^{\tilde{t}_2} (\dot{\Pi} - \dot{T} + \dot{V}) d\tilde{t} = 0, \quad (2.8)$$

In Eq. (2.8), variables  $\Pi$ ,  $V$ , and  $T$  are the strain energy, work done, and kinetic energy, respectively. Hence, the virtual strain energy can be given as follows:

$$\delta \dot{\Pi} = \int_V (\sigma_{ij} \delta \varepsilon_{ij} dV - D_x E_x - D_z E_z) dz dx, \quad (2.9)$$

$$\phi(x, z, \tilde{t}) = -\cos(\beta z) \phi(x, \tilde{t}) + \frac{2zV_0}{h} e^{-ik\tilde{t}}, \quad (2.10)$$

where  $D_x = e_{15} \gamma_{xz} + e_{11} E_x$  and  $D_z = e_{31} \varepsilon_{xx} + e_{33} E_z$ . The variation in electric potential in the  $x$  direction is  $\beta = \pi/h$ ;  $\phi(x, \tilde{t})$ ;  $V_0$  and  $\Omega$  are the external electric voltage and natural frequency of the piezoelectric nanobeam, respectively.

By infusing Eq. (2.6) in Eq. (2.9),

$$\begin{aligned}\delta \Pi &= \int_0^L \left( N \frac{\partial \delta \Pi}{\partial x} - M_b \frac{\partial^2 \delta w_b}{\partial x^2} - M_s \frac{\partial^2 \delta w_s}{\partial x^2} \right. \\ &\quad \left. + Q \frac{\partial \delta w_s}{\partial x} - D_x E_x - D_z E_z \right) dz dx,\end{aligned}\quad (2.11)$$

and the stress resultants can be obtained as follows:

$$[N, M_b, M_s] = \int_A [1, z, f(z)] \sigma_{xx} dA, \quad (2.12)$$

$$Q = \int_A g(z) \sigma_{xz} dA, \quad (2.13)$$

The kinetic energy of the system can be determined as follows:

$$\delta T = \int_V \rho(z) \left( \frac{\partial \Pi_x}{\partial \tilde{t}} \frac{\partial \delta \Pi_x}{\partial \tilde{t}} + \frac{\partial \Pi_z}{\partial \tilde{t}} \frac{\partial \delta \Pi_z}{\partial \tilde{t}} \right) dV, \quad (2.14)$$

Infusion of Eq. (2.4) in Eq. (2.14) results in the following:

$$\delta T = \int_0^L \left( \begin{aligned} & I_0^* \left( \frac{\partial \Pi}{\partial \tilde{t}} \frac{\partial \delta \Pi}{\partial \tilde{t}} + \frac{\partial (w_b + w_s)}{\partial \tilde{t}} \frac{\partial \delta (w_b + w_s)}{\partial \tilde{t}} \right) \\ & - I_1^* \left( \frac{\partial \Pi}{\partial \tilde{t}} \frac{\partial^2 \delta w_b}{\partial x \partial \tilde{t}} + \frac{\partial^2 w_b}{\partial x \partial \tilde{t}} \frac{\partial \delta \Pi}{\partial \tilde{t}} \right) \\ & - J_1^* \left( \frac{\partial \Pi}{\partial \tilde{t}} \frac{\partial^2 \delta w_s}{\partial x \partial \tilde{t}} + \frac{\partial^2 w_s}{\partial x \partial \tilde{t}} \frac{\partial \delta \Pi}{\partial \tilde{t}} \right) \\ & + I_2^* \frac{\partial^2 w_b}{\partial x \partial \tilde{t}} \frac{\partial^2 \delta w_b}{\partial x \partial \tilde{t}} + K_2^* \frac{\partial^2 w_s}{\partial x \partial \tilde{t}} \frac{\partial^2 \delta w_s}{\partial x \partial \tilde{t}} \\ & + J_2^* \left( \frac{\partial^2 w_b}{\partial x \partial \tilde{t}} \frac{\partial^2 \delta w_s}{\partial x \partial \tilde{t}} + \frac{\partial^2 w_s}{\partial x \partial \tilde{t}} \frac{\partial^2 \delta w_b}{\partial x \partial \tilde{t}} \right) \end{aligned} \right), \quad (2.15)$$

In accordance with the magnetic and temperature effect,

$$f_B = \eta A \Omega_x^2 \frac{\partial^2 w}{\partial x^2} \quad N_T = \int_{-h/2}^{h/2} E(z) \lambda(z) \Delta T dz,$$

where  $f_B$ ,  $\eta$ ,  $A$ , and  $\Omega_x$  stand for the magnetic force, magnetic field permeability, cross sectional area of the nano-beam, and the magnetic potential of the longitudinal magnetic field. For an FG nanobeam, it is assumed that the temperature can be distributed uniformly across its thickness and the temperature gradient at stress free state is  $\Delta T$ .

In the above definition, the inertia of mass moments can be defined as follows:

$$[I_0^*, I_1^*, I_2^*, J_1^*, J_2^*, K_2^*] = \int_A [1, z, z^2, f(z), zf(z), f^2(z)] \rho(z) dA$$

and the work done  $N_x$  with a temperature gradient of thermal effect can be defined as follows:

$$\delta V = \left( \frac{1}{2} \int_0^l N_x + N_T \left( \frac{\partial w}{\partial x} \frac{\partial \delta w}{\partial x} \right) + \eta \Omega_x^2 \frac{\partial^2 w}{\partial x^2} \right) dx, \quad (2.16)$$

By inserting the Eqs. (2.11) and (2.15) in Eq. (2.8), the equation of the beam in Euler–Lagrange can be derived and the outcome can be coupled as follows:

$$\frac{\partial N}{\partial x} = I_0^* \frac{\partial^2 \Pi}{\partial \tilde{t}^2} - I_1^* \frac{\partial^3 w_b}{\partial x \partial \tilde{t}^2} - J_1^* \frac{\partial^3 w_s}{\partial x \partial \tilde{t}^2}, \quad (2.17)$$

$$\begin{aligned} \frac{\partial^2 M_b}{\partial x^2} &= I_0^* \frac{\partial^2 (w_b + w_s)}{\partial \tilde{t}^2} + I_1^* \frac{\partial^3 \Pi}{\partial x \partial \tilde{t}^2} - I_2^* \frac{\partial^4 w_b}{\partial x^2 \partial \tilde{t}^2} \\ &- J_2^* \frac{\partial^4 w_s}{\partial x^2 \partial \tilde{t}^2} - \left( N_x + N_T \left( \frac{\partial^2 w}{\partial x^2} + \eta \Omega_x^2 \frac{\partial^2 w}{\partial x^2} \right) \right), \end{aligned} \quad (2.18)$$

$$\begin{aligned} \frac{\partial^2 M_b}{\partial x^2} + \frac{\partial Q}{\partial x} - N_x \frac{\partial^2 w}{\partial x^2} \\ = \left( I_0^* \frac{\partial^2 (w_b + w_s)}{\partial \tilde{t}^2} + J_1^* \frac{\partial^3 \Pi}{\partial x \partial \tilde{t}^2} - J_2^* \frac{\partial^4 w_b}{\partial x^2 \partial \tilde{t}^2} \right. \\ \left. - K_2^* \frac{\partial^4 w_s}{\partial x^2 \partial \tilde{t}^2} \right) - \left( N_x + N_T \left( \frac{\partial^2 w}{\partial x^2} + \eta \Omega_x^2 \frac{\partial^2 w}{\partial x^2} \right) \right), \end{aligned} \quad (2.19)$$

$$\delta \phi = \int_{-\frac{h}{2}}^{\frac{h}{2}} \left[ \cos(\beta z) \left( \frac{\partial D x}{\partial x} \right) \right] + \beta \sin(\beta z) D z. \quad (2.20)$$

### 3 Nonlocal state-space model

This section demonstrates the nonlocality stress and strain effects on the time–space domains, when the wave length or excitation frequency interferes with time and intrinsic characteristic length. Nonlocal time–space viscoelasticity problems are based on the combination of the Boltzmann superposition integral and the Eringen concept of nonlocal elasticity. Accordingly, integral stress in nonlinear state and strain equations are stated as follows:

$$\begin{aligned} \int_{-\infty}^t \int_{-\infty}^v (K_\sigma(\tilde{t} - \tau, |r - r'|) \sigma_{ij}(r', \tau) dr' d\tau) \\ = \int_{-\infty}^t \int_{-\infty}^v K_\epsilon(\tilde{t} - \tau, |r - r'|) C_{ijkl} \epsilon_{kl}(r', \tau) dr' d\tau, \end{aligned} \quad (3.1)$$

where  $\sigma_{ij}$  and  $\epsilon_{kl}$  are the stress and strain tensor arrays and the nonlocal kernel functions are  $K_\sigma(\tilde{t} - \tau, |r - r'|)$  and  $K_\epsilon(\tilde{t} - \tau, |r - r'|)$ . Eq. (3.1) can be read in the following form *via* Fourier, inverse Fourier, and Taylor series,

$$\left( 1 - l_\sigma^2 \nabla^2 + \tau_\sigma \frac{\partial}{\partial \tilde{t}} \right) \sigma_{ij} = C_{ijkl} \left( 1 - l_\epsilon^2 \nabla^2 + \tau_\epsilon \frac{\partial}{\partial \tilde{t}} \right) \epsilon_{kl}, \quad (3.2)$$

To balance the absence of stiffness-hardening behavior, the nonlocal strain gradient elasticity must be incorporated in the equation. The following relation can be used to derive the nonlocal strain gradient viscoelasticity with fraction.

$$\left( 1 - l_\sigma^2 \nabla^2 + \tau_\sigma \frac{\partial}{\partial \tilde{t}} \right) \sigma_{ij} = C_{ijkl} \left( 1 - l_\epsilon^2 \nabla^2 + \tau_\epsilon \frac{\partial}{\partial \tilde{t}} \right) \epsilon_{kl}, \quad (3.3)$$

The Kelvin–Voigt relation of viscoelastic material with three parameters in a solid state is given as follows:

$$(1 - \mu^2 \nabla^2) \sigma_{ij} = C_{ijkl} \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \varepsilon_{kl}, \quad (3.4)$$

where  $\lambda = l_e$  and  $\mu = l_\sigma$  are length scale and nonlocal parameters, respectively. The relation between rheological character and spatial nonlocality of the system is designed as  $\tau = \mu \sqrt{\frac{\tau_c}{E_c}}$ . From Eqs. (2.12) and (2.13), the displacement field having resultant stress over the area of cross section of the beam can be computed as follows:

$$(1 - \mu^2 \nabla^2) N = \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \times \left( A_{xx} \frac{\partial u}{\partial x} - B_{xx} \frac{\partial^2 w_b}{\partial x^2} - B_{xx}^s \frac{\partial^2 w_s}{\partial x^2} \right) - e_{31} E_z, \quad (3.5)$$

$$(1 - \mu^2 \nabla^2) M^b = \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \times \left( B_{xx} \frac{\partial u}{\partial x} - D_{xx} \frac{\partial^2 w_b}{\partial x^2} - D_{xx}^s \frac{\partial^2 w_s}{\partial x^2} \right) - (N_x + N_T - \eta \Omega_x^2) \frac{\partial^2 w}{\partial x^2} - e_{31} E_z, \quad (3.6)$$

$$(1 - \mu^2 \nabla^2) M^s = \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \times \left( B_{xx}^s \frac{\partial u}{\partial x} - D_{xx}^s \frac{\partial^2 w_b}{\partial x^2} - H_{xx}^s \frac{\partial^2 w_s}{\partial x^2} \right) - (N_x + N_T - \eta \Omega_x^2) \frac{\partial^2 w}{\partial x^2} - e_{31} E_z, \quad (3.7)$$

$$(1 - \mu^2 \nabla^2) Q^{xz} = \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \times \left( A_{xx}^s \frac{\partial w_s}{\partial x} \right) - (N_x + N_T - \eta \Omega_x^2) \frac{\partial^2 w}{\partial x^2} - e_{15} E_x, \quad (3.8)$$

$$(1 - \mu^2 \nabla^2) D_x = e_{15} \gamma_{xz} + e_{11} E_x, \quad (3.9)$$

$$(1 - \mu^2 \nabla^2) D_z = e_{31} \varepsilon_{xx} + e_{33} E_z, \quad (3.10)$$

where cross sectional rigidities are

$$[A_{xx}, B_{xx}, B_{xx}^s, D_{xx}, D_{xx}^s, H_{xx}^s] = \int_A [1, z, f(z), z^2, zf(z), f^2(z)], \quad (3.11)$$

$$A^s = \int_A g^2(z) G(z) dA, \quad (3.12)$$

where  $e_{31} E_x = e_{31} (\cos(\beta z)) \frac{\partial \phi}{\partial x}$  and  $e_{15} E_z = e_{15} (-\beta \sin(\beta z))$

$$\phi - \frac{2V_0 e^{ikt}}{h}.$$

## 4 Governing equations

Eqs. (3.5)–(3.10) must be substituted in equations (2.17)–(2.20). Now the governing equations are as follows:

$$\begin{aligned} & \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \left( A_{xx} \frac{\partial^2 \Pi}{\partial x^2} - B_{xx} \frac{\partial^3 w_b}{\partial x^3} - B_{xx}^s \frac{\partial^3 w_s}{\partial x^3} \right) \\ & + (1 - \mu^2 \nabla^2) \left( -I_0 \ddot{\Pi} + I_1 \frac{\partial \dot{w}_b}{\partial x} + J_1 \frac{\partial \dot{w}_s}{\partial x} \right) \\ & - \left( N_x + N_T - \eta \Omega_x^2 \left( \frac{\partial^2 w}{\partial x^2} \right) \right) - e_{31} E_z = 0, \end{aligned} \quad (4.1)$$

$$\begin{aligned} & \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \left( B_{xx} \frac{\partial^3 \Pi}{\partial x^3} - D_{xx} \frac{\partial^4 w_b}{\partial x^4} - D_{xx}^s \frac{\partial^4 w_s}{\partial x^4} \right) \\ & + (1 - \mu^2 \nabla^2) \left( -I_0 (\ddot{w}_b + \ddot{w}_s) - I_1 \frac{\partial \ddot{\Pi}}{\partial x} + I_2 \frac{\partial^2 \dot{w}_b}{\partial x^2} \right. \\ & \left. + J_2 \frac{\partial^2 \dot{w}_s}{\partial x^2} \right) - \left( N_x + N_T - \eta \Omega_x^2 \left( \frac{\partial^2 w}{\partial x^2} \right) \right) - e_{31} E_z = 0, \end{aligned} \quad (4.2)$$

$$\begin{aligned} & \left( 1 - \lambda^2 \nabla^2 + \tau \frac{\partial}{\partial t} \right) \left( B_{xx}^s \frac{\partial^3 \Pi}{\partial x^3} - D_{xx}^s \frac{\partial^4 w_b}{\partial x^4} \right. \\ & \left. - H_{xx}^s \frac{\partial^4 w_s}{\partial x^4} + A^s \frac{\partial^2 w_s}{\partial x^2} \right) + (1 - \mu^2 \nabla^2) \left( -I_0 (\ddot{w}_b + \ddot{w}_s) \right. \\ & \left. - J_1 \frac{\partial \ddot{\Pi}}{\partial x} + J_2 \frac{\partial^2 \dot{w}_b}{\partial x^2} + K_2 \frac{\partial^2 \dot{w}_s}{\partial x^2} \right) \\ & - \left( N_x + N_T - \eta \Omega_x^2 \left( \frac{\partial^2 w}{\partial x^2} \right) \right) - e_{31} E_z - e_{15} E_x = 0. \end{aligned} \quad (4.3)$$

## 5 Analytical solution

The governing equation obtained in Section 4 will be solved in this section. The analytical wave dispersion method is used in this case to solve the problems of wave propagation for various types of structures, including beams, plates, and shells. A higher order beam's solution function can be in the form as follows:

$$\begin{Bmatrix} \Pi \\ w_b \\ w_s \end{Bmatrix} = \begin{Bmatrix} \Pi \exp[i(\beta x - \omega t)] \\ w_b \exp[i(\beta x - \omega t)] \\ w_s \exp[i(\beta x - \omega t)] \end{Bmatrix}, \quad (5.1)$$

where  $\Pi$ ,  $w_b$ , and  $w_s$  are the anonymous amplitudes of propagating waves. Here  $\omega$  &  $\beta$  are frequency and wave number, respectively. By substituting the above expression in Eqs. (4.1)–(4.3), the achieved form will be as follows:

$$\{[K] + [C]\omega + [M]\omega^2\} \begin{Bmatrix} \Pi \\ w_b \\ w_s \end{Bmatrix} = 0, \quad (5.2)$$

where damping and mass matrices are given by  $[K]$ ,  $[C]$ , and  $[M]$  respectively. The components of these symmetric matrices are as follows:

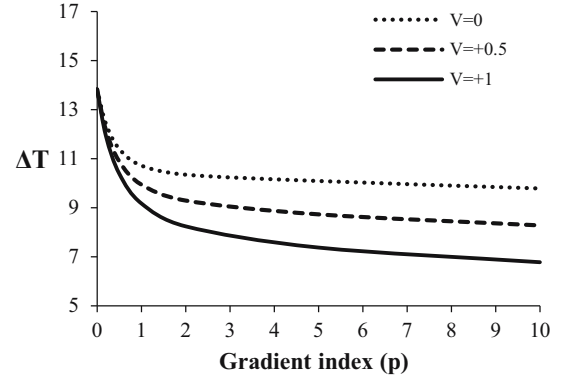
$$\begin{aligned} k_{11} &= (1 + \lambda^2\beta^2)A_{xx}\beta^2 \\ k_{12} &= i(N_x + \eta\Omega_x^2)(1 + \lambda^2\beta^2)B_{xx}\beta^5 \\ k_{13} &= i(N_x + \eta\Omega_x^2)(1 + \lambda^2\beta^2)B_{xx}^s\beta^5 \\ k_{14} &= e_{31}(1 - \lambda^2\beta^2)\beta \sin(\beta z)\phi - \frac{2V_0e^{ikt}}{h} \\ k_{22} &= -(N_x + \eta\Omega_x^2)N_T(1 + \lambda^2\beta^2)D_{xx}\beta^6 \\ k_{23} &= -(N_x + \eta\Omega_x^2)N_T(1 + \lambda^2\beta^2)D_{xx}^s\beta^6 \\ k_{24} &= e_{31}(1 - \lambda^2\beta^2)\beta \sin(\beta z)\phi - \frac{2V_0e^{ikt}}{h} \\ k_{33} &= -(N_x + \eta\Omega_x^2)N_T(1 + \lambda^2\beta^2)(H_{xx}^s\beta^6 + A^s\beta^4) \\ k_{34} &= e_{31}(1 - \lambda^2\beta^2)\beta^2 \sin(\beta z)\phi - \frac{2V_0e^{ikt}}{h}, \end{aligned} \quad (5.3)$$

$$\begin{aligned} c_{11} &= -A_{xx}\tau\beta^2 \\ c_{12} &= i(N_x + \eta\Omega_x^2)B_{xx}\tau\beta^5 \\ c_{13} &= i(N_x + \eta\Omega_x^2)B_{xx}^s\tau\beta^5 \\ c_{22} &= -(N_x + \eta\Omega_x^2)D_{xx}\tau\beta^6 \\ c_{23} &= -(N_x + \eta\Omega_x^2)D_{xx}^s\tau\beta^6 \\ c_{24} &= \tau e_{31} \left( \beta^2 \cos(\beta z) \frac{dz}{dt} \phi - \left( \frac{2V_0e^{ikt}}{h} (ik) \right) \right) \\ c_{33} &= -(N_x + \eta\Omega_x^2)\tau\beta^4 (H_{xx}^s\beta^2 + A_s) \\ c_{34} &= \tau e_{31} \left( \beta^2 \cos(\beta z) \frac{dz}{dt} \phi - \left( \frac{2V_0e^{ikt}}{h} (ik) \right) \right), \end{aligned} \quad (5.4)$$

$$\begin{aligned} m_{11} &= -(1 + \mu^2\beta^2)I_0^* \\ m_{12} &= i(N_x + \eta\Omega_x^2)\beta^2(1 + \mu^2\beta^2)I_1^* \\ m_{13} &= i(N_x + \eta\Omega_x^2)\beta^2(1 + \mu^2\beta^2)J_1^* \\ m_{22} &= -(1 + \mu^2\beta^2)(I_0^* + I_2^*(N_x + \eta\Omega_x^2)\beta^4) \\ m_{23} &= -(1 + \mu^2\beta^2)(I_0^* + J_2^*(N_x + \eta\Omega_x^2)\beta^4) \\ m_{33} &= -(1 + \mu^2\beta^2)(I_0^* + K_2^*(N_x + \eta\Omega_x^2)\beta^2) \\ m_{34} &= (1 + \mu^2\beta^2)e_{31} \left( \beta^2 \cos(\beta z) \frac{dz}{dt} \phi - \left( \frac{2V_0e^{ikt}}{h} (ik) \right) \right). \end{aligned} \quad (5.5)$$

## 6 Results and discussion

This section illustrates the magneto-thermo vibration of FG nanobeam with the numerical examples. The material



**Figure 2:** External voltage with the presence of rising temperature to gradient index for  $\mu = 1.0$ .

properties are composed of  $\text{BaTiO}_3$  and  $\text{CoFe}_2\text{O}_4$ , which are presented in Table 1. Table 2 shows the comparison of the buckling load for various power-law exponent values. Whereas, Table 3 gives the variation in the magnetic potential and electric voltages with the varying and increasing nonlocal parameter. Since an increasing nonlocal parameter shows a decrement with the magnetic potential ( $\Omega$ ) and the electric voltage ( $V$ ).

Now, Figures 2 and 3 highlight the effect of external voltage ( $V$ ) with the variation in the rising temperature ( $\Delta T$ ) and the gradient index ( $p$ ) for  $\mu = 1.0$  &  $1.5$ . Whereas, with the increase in the external voltage ( $V = 0, 0.5, \text{ and } 1$ ), the temperature gradually decreases with respect to the gradient index ( $p$ ) and nonlocal parameter. Figure 4 shows the decrement in the buckling load with an increase in the magnetic effect ( $\Omega$ ) and the nonlocal value ( $\mu$ ). Also, when there is no magnetic effect found, the buckling load at some point decreases with the increase in the value of the nonlocal ( $\mu$ ). Figure 5 presents the variation in the dimensionless buckling load with the nonlocal parameter ( $\mu$ ) in the effect of the external voltage ( $V$ ). Since there is an increase in the voltage ( $V$ ), the dimensionless buckling load declines and gets neutralized at some point of the nonlocal parameter. Figures 6 and 7 depict the effect of gradient index with respect to

**Table 1:** MEE coefficients of material properties

Material	Properties			
$\text{BaTiO}_3$	$E$ (Pa)	166	$e_{33}$ (N/m <sup>2</sup> K)	$(7.124 \times 10^{-9})$
	$\rho$ (kg/m <sup>3</sup> )	5,800	$e_{15}$ (c/m <sup>2</sup> )	14.1
	$\nu$ (-)	1.1945		
$\text{CoFe}_2\text{O}_4$	$E$ (Pa)	286	$e_{31}$ (c/m <sup>2</sup> )	$-(4.1)$
	$\rho$ (kg/m <sup>3</sup> )	5,300	$e_{11}$ (c/V m)	$(5.841 \times 10^{-9})$
	$\nu$ (-)	1.167		

**Table 2:** Comparison of FGM beam non-dimensional buckling for various power-law exponents

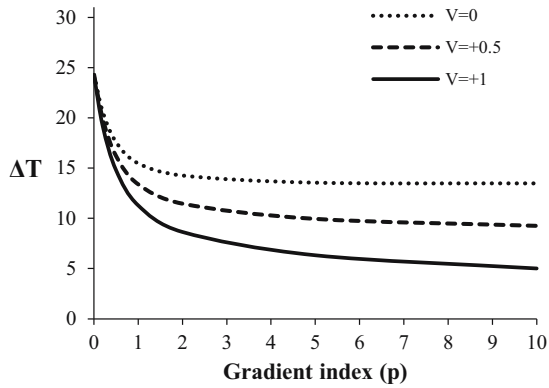
$L/h$		$p = 0$	$p = 0.5$	$p = 1$	$p = 2$	$p = 5$	$p = 10$
5	Nguyen <i>et al.</i> (2015)	48.8406	32.0013	24.6894	19.1577	15.7355	14.1448
	Present	48.835	31.967	24.6870	19.1605	15.7401	14.13
10	Nguyen <i>et al.</i> (2015)	52.3083	34.0002	26.1707	20.3909	17.1091	15.5278
	Present	52.3082	34.0087	26.1727	20.3936	17.1118	15.5291

**Table 3:** Dimensionless frequency of an FG nanobeam varies with nonlocal parameters, electric voltages, and magnetic potentials

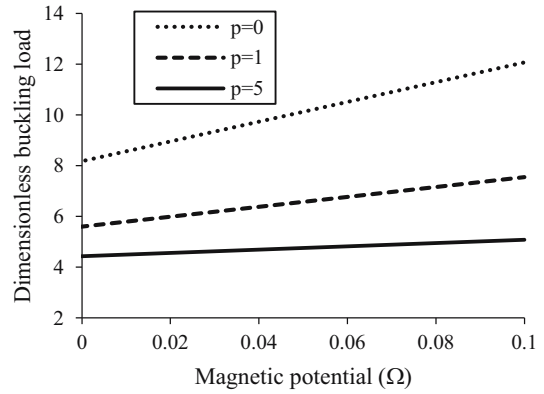
$\mu$		$p = 0.2$			$p = 1$			$p = 5$		
		$V = -5$	$V = 0$	$V = +5$	$V = -5$	$V = 0$	$V = +5$	$V = -5$	$V = 0$	$V = +5$
0	$\Omega = -0.05$	8.34927	7.68726	7.24086	8.3637	7.69636	7.24398	8.3781	7.70545	7.24711
	$\Omega = 0$	8.34708	7.68034	7.22898	8.36151	7.68946	7.23212	8.37592	7.69856	7.23525
	$\Omega = -0.05$	8.34489	7.67343	7.21709	8.35933	7.68255	7.22023	8.37373	7.69166	7.22337
1	$\Omega = -0.05$	7.96429	7.33365	6.9088	7.97941	7.34319	6.91208	7.9945	7.35273	6.91536
	$\Omega = 0$	7.96199	7.32641	6.89636	7.97712	7.33596	6.89964	7.99222	7.3455	6.90293
	$\Omega = -0.05$	7.95969	7.31915	6.88389	7.97483	7.32872	6.88718	7.98993	7.33827	6.89047
2	$\Omega = -0.05$	7.62789	7.02471	6.61874	7.64368	7.03468	6.62216	7.65944	7.04462	6.62558
	$\Omega = 0$	7.6255	7.01715	6.60575	7.64129	7.02712	6.60917	7.65705	7.03708	6.6126
	$\Omega = -0.05$	7.6231	7.00958	6.59273	7.6389	7.01956	6.59617	7.65466	7.02953	6.5996
3	$\Omega = -0.05$	7.33065	6.75176	6.3625	7.34708	6.76213	6.36606	7.36347	6.77248	6.36962
	$\Omega = 0$	7.32816	6.74389	6.34898	7.34459	6.75427	6.35255	7.36099	6.76463	6.35612
	$\Omega = -0.05$	7.32566	6.73601	6.33544	7.3421	6.7464	6.33902	7.3585	6.75678	6.34259
0	$\Omega = -0.05$	8.55703	8.31260	8.37748	9.43205	8.84645	8.55259	10.2325	9.34987	8.72418
	$\Omega = 0$	8.41640	7.88376	7.67745	9.30465	8.44476	7.86815	10.1152	8.97075	8.05433
	$\Omega = -0.05$	8.27337	7.43021	6.90683	9.17548	8.02299	7.11820	9.99651	8.57489	7.32348
1	$\Omega = -0.05$	8.08984	7.91790	8.03872	9.01035	8.47664	8.22104	9.84516	9.00077	8.39941
	$\Omega = 0$	7.94094	7.46642	7.30630	8.87690	8.05654	7.50644	9.72317	8.60629	7.70137
	$\Omega = -0.05$	7.78919	6.98583	6.49177	8.74141	7.61329	6.71622	9.59964	8.19284	6.93341
2	$\Omega = -0.05$	7.67792	7.57252	7.74446	8.64241	8.15496	7.93354	9.50958	8.69849	8.11823
	$\Omega = 0$	7.52087	7.09911	6.98123	8.50319	7.71737	7.19042	9.38323	8.28964	7.39369
	$\Omega = -0.05$	7.36046	6.59180	6.12362	8.36165	7.25342	6.36107	9.25516	7.85955	6.58997
3	$\Omega = -0.05$	7.31058	7.26690	7.48595	8.31777	7.87198	7.68141	9.21554	8.43377	7.87201
	$\Omega = 0$	7.14545	6.77216	6.69332	8.17302	7.41771	6.91123	9.08510	8.01142	7.12247
	$\Omega = -0.05$	6.97642	6.23831	5.79324	8.02566	6.93375	6.04369	8.95277	7.56553	6.28416

the presence of the dimensionless buckling load and magnetic potential for different nonlocal values. Therefore, with the surge effect of the gradient index ( $p$ ), there is an increase in the magnetic potential with the decrease in the dimensionless buckling load and amplifies over the nonlocal parameter ( $\mu$ ). Figures 8 and 9 represent the impact of the electric voltage and dimensionless buckling load in the presence of the gradient index  $p = 0, 1, 5$  over

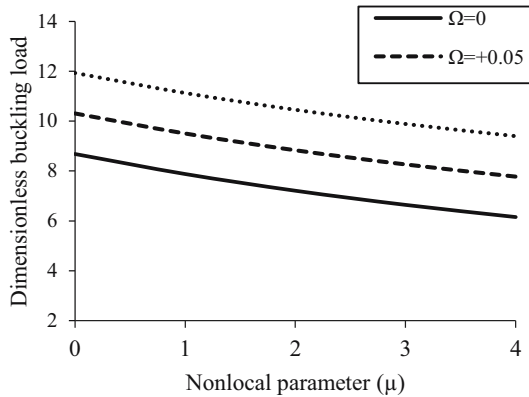
nonlocal parameter ( $\mu$ ). Hence, when the gradient index is null, there is no fluctuation in the buckling load and forms a linear effect and nonlocal values amplify the buckling load. So, if the gradient index rises, then the electric voltage increases with the decrease in buckling load gradually. Figures 10 and 11 interpret that temperature ( $\Delta T$ ) stabilizes with an increase in the gradient index and magnetic potential. Thus, when the magnetic potential  $\Omega = 0$ ,



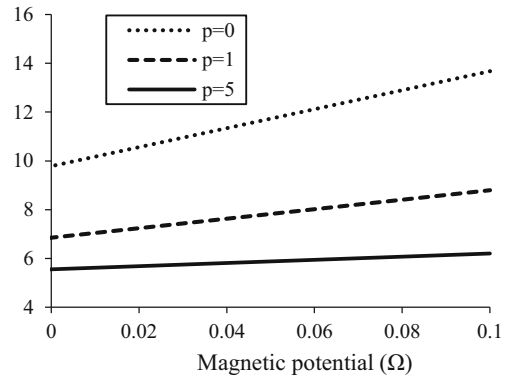
**Figure 3:** External voltage in the presence of temperature with respect to gradient index for  $\mu = 1.5$ .



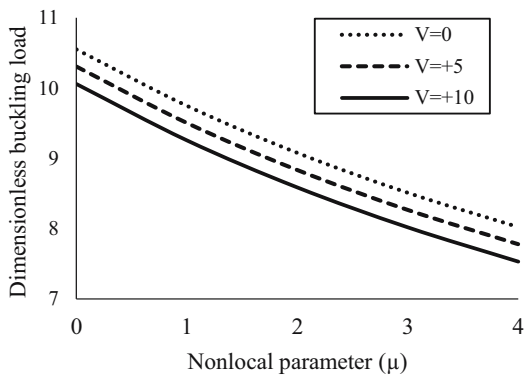
**Figure 6:** Gradient index with the presence of rising dimensionless buckling load to magnetic potential for  $\mu = 1.0$ .



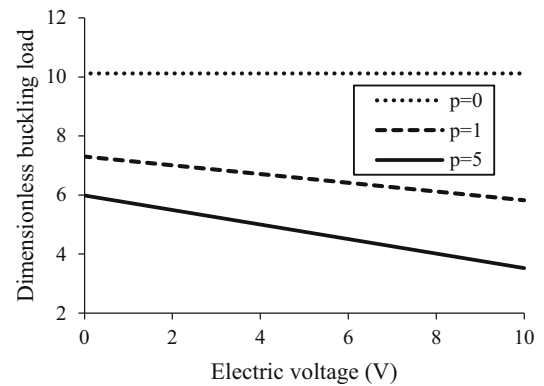
**Figure 4:** Magnetic potential on the dimensionless buckling with respect to nonlocal parameter.



**Figure 7:** Gradient index in the presence of dimensionless buckling load to magnetic potential  $\mu = 1.5$ .

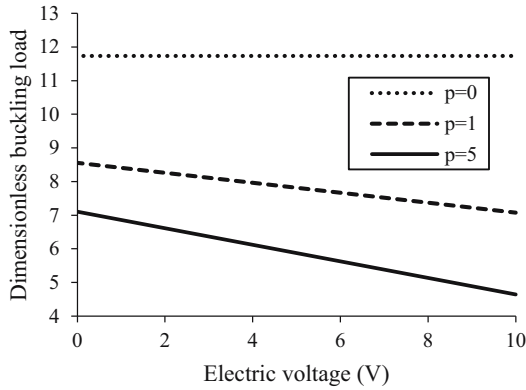


**Figure 5:** External voltage on the dimensionless buckling load with respect to nonlocal parameter.

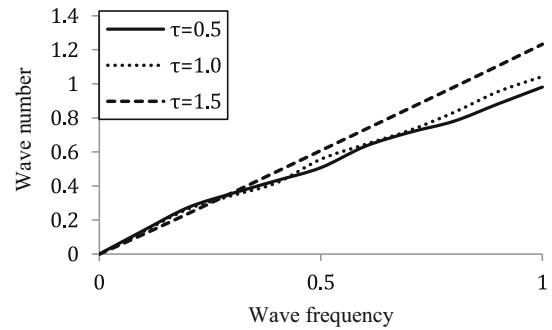


**Figure 8:** Gradient index in the presence of dimensionless buckling load to electric voltage when  $\mu = 1.0$ .

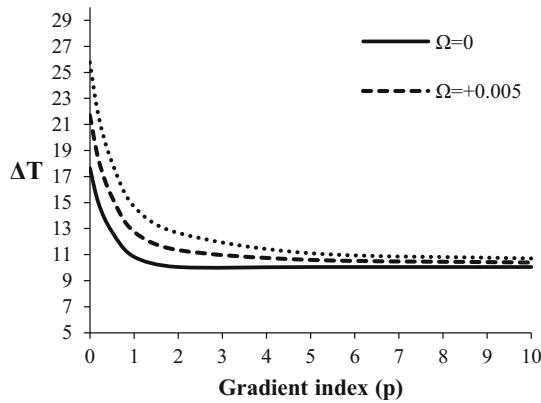




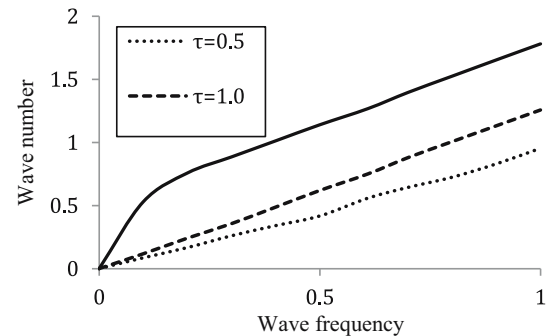
**Figure 9:** Gradient index in the presence of dimensionless buckling load to electric voltage when  $\mu = 1.5$ .



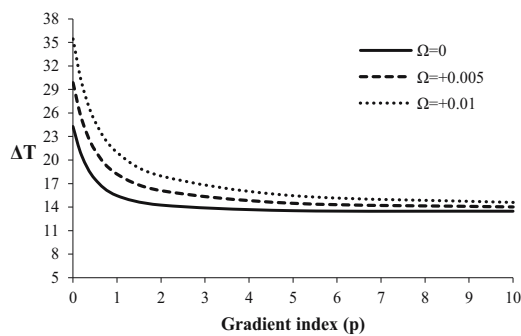
**Figure 12:** Damping factor in the presence of wave number with respect to wave frequency for  $\mu = 1.0$ .



**Figure 10:** Magnetic potential in the presence of temperature with respect to gradient index for  $\mu = 1.0$ .



**Figure 13:** Damping factor in the presence of wave number with respect to wave frequency for  $\mu = 1.5$ .



**Figure 11:** Magnetic potential in the presence of temperature with respect to gradient index for  $\mu = 1.5$ .

the temperature calms down and stabilizes at some point with respect to an increase in the gradient index ( $p$ ) and nonlocal values. Figures 12 and 13 show the effect of an internal damping factor ( $\tau$ ) in the existence of wave number and wave frequency over nonlocal parameter ( $\mu$ ).

In these figures, the damping factor with  $\tau \geq 1.5$  shows a raising linear value and when  $\tau \leq 0.5$ , there is an oscillation in the wave number and wave frequency.

## 7 Conclusion

The above study shows the wave propagation analysis of piezoelectric FGM nanobeam. Magneto thermo properties of the FGM nanobeam are considered to be the function of thickness according to the power-law model. The governing equations are extracted by substituting the structure displacement field equations in the beam's Euler-Lagrange equations and are framed as symmetric matrices components to arrive at required solutions. Hence, the upshots of the work are as follows:

- The stability behaviors of FGM nanobeam are affected by magneto thermo piezo electricity and nonlocal values.

- Physical variants could be controlled *via* applying a suitable value of damping factor.
- Natural frequency reduces, while the nonlocal parameter and gradient index of the FG nanobeam amplify.
- The increase in power-law index softens the volume fraction.
- The bending rigidity and phase velocities are high in amplified wave numbers and get reversed in low wave numbers.

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**Data availability statement:** The datasets analysed during the current study are available from the corresponding author on reasonable request.

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